

# The Restoring Force

Hooke's Law and the Birth of the Science of Materials

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# Chapter 1

## The World Before Elasticity

### Learning Objectives

- Understand what engineers and scientists knew about structural behavior before Hooke's Law
- Recognize why Galileo's pioneering beam analysis was correct in approach but wrong in detail
- See how the absence of a deformation law made structural design a matter of guesswork and costly experience

### 1.1 Galileo at Padua

In the winter of 1638, Galileo Galilei sat under house arrest in Arcetri, half-blind, forbidden from publishing, and composed one of the most consequential works in the history of engineering. He was seventy-four years old. The book was called *Discorsi e Dimostrazioni Matematiche intorno a Due Nuove Scienze* — *Discourses and Mathematical Demonstrations Relating to Two New Sciences* — and it contained, buried in its second day's dialogue, the first serious mathematical attempt to understand why structures break.

Galileo had watched beams fail. He had seen the workmen in the Venetian Arsenal snap oars and crack spars, and he understood intuitively that a longer beam required a thicker cross-section to carry the same load. This was ancient practical knowledge. What Galileo wanted was a *reason* — a mathematical account that would let an engineer calculate, not merely estimate. He chose the cantilever beam as his test case: a horizontal rod fixed at one end, loaded at the other, and threatening to snap at the wall.

The picture he drew was right. He understood that the beam's outer fibers — the ones farthest from the neutral axis — were being stretched on the top and compressed on the bottom (for a downward load). He correctly identified the fixed end as the location of maximum stress. These insights were not trivial. They had eluded mathematicians for centuries, and they remain the foundation of every structural analysis performed today.

But Galileo got the stress distribution wrong, and the error was not a small one. He assumed that all of the tensile resistance was concentrated at the bottom fiber of the beam — the extreme

edge — rather than distributed across the cross-section in proportion to distance from the neutral axis. In modern language, he placed the neutral axis at the bottom of the beam instead of at its centroid. The consequence was that his formula for the breaking load was off by a factor of three, though in the wrong direction: he overestimated the strength of beams, which is a particularly dangerous mistake for an engineer to make.

## 1.2 The Problem Galileo Was Solving

To appreciate what Galileo was attempting, it helps to understand the practical stakes. The seventeenth century was building at a scale that previous generations had not attempted. Fortifications were growing larger. Ships were growing heavier. The cannon — that indiscriminate destroyer of both armies and the cannons themselves, which had an unfortunate habit of bursting at the breech — was demanding materials analysis that nobody knew how to provide.

When a cannon burst, or a deck beam broke, or a masonry arch collapsed, the engineers responsible had no theory to consult. They had rules of thumb — accumulated through generations of trial and considerable error — but they had no principled account of *why* things broke or how to predict whether a given design would fail. Stronger materials helped, but the margin between strong enough and catastrophically weak was a matter of experience, not calculation.

Galileo saw this problem clearly. A larger ship needed larger timbers, but in what proportion? If you doubled the length of a beam, did you double its cross-section or quadruple it? The answer depended on the relationship between load, geometry, and material strength, and in 1638 nobody had that relationship in mathematical form.

His approach was characteristically bold. He treated the beam as a lever, with the fulcrum at the base of the fixed end, and the tensile force in the beam's material acting as the resistance arm. The geometry was compelling and not entirely wrong. The problem was that his fulcrum assumption predetermined the stress distribution, and the stress distribution he assumed bore no resemblance to the actual distribution that a material obeying Hooke's Law would produce.

## 1.3 What Was Missing

The failure in Galileo's analysis was not a mathematical error. The algebra was sound. The failure was conceptual: he had no law relating the deformation of a material to the force applied to it.

Without such a law, he could not determine how the internal stresses were distributed across the cross-section of a beam. He knew that the beam resisted the load somehow, but he could not say *where* — inside the material, distributed across the fibers — the resistance resided. His guess, placing all resistance at the outermost fiber, was the simplest possible assumption. It was also, as we now know, incorrect by a significant margin.

This missing piece was not obscure. It was simply unknown. No one, in 1638, had a quantitative description of how materials deform under load. Springs were known to be elastic — they returned to their original shape — but the relationship between the force applied and the amount of deformation had not been measured, stated, or used in calculation. The very concept of a material property relating force to deformation did not yet exist.

Galileo was, in a sense, trying to solve a problem whose key ingredient had not yet been discovered. It is the intellectual equivalent of deriving the pressure in a gas before anyone had stated the ideal gas law: the framework is there, the reasoning is there, but the crucial proportionality is absent.

## 1.4 Mariotte and the Missing Step

In 1680, four decades after Galileo and two years after Hooke, the French physicist Edme Mariotte published *Traité du Mouvement des Eaux et des Autres Corps Fluides*, which contained, almost incidentally, a correction to Galileo's beam analysis. Mariotte realized that the neutral axis of a bent beam lies at its centroid — the geometric center — not at its bottom edge. His physical intuition was better: he recognized that fibers above the neutral axis are in tension and fibers below are in compression, and that neither tension nor compression is uniformly distributed but varies linearly across the cross-section.

Mariotte was right about the neutral axis. But he was still working without Hooke's Law, which meant he was working without a rigorous foundation for the linear stress distribution he was assuming. He arrived at the correct qualitative picture through physical intuition. The mathematical confirmation — the proof that an elastic material following Hooke's Law *must* produce a linear stress distribution across a bent beam — would come a century later, after the law had been discovered, named, and gradually extended from springs to solids.

The lesson of this interlude is important. In the seventeenth century, there were brilliant people trying to do structural mechanics. Galileo, Mariotte, Christiaan Huygens, and others were attacking these problems with considerable sophistication. What stopped them was not a failure of intelligence or method. It was the absence of a physical law that no one had yet stated.

That law was being worked out, at almost exactly this moment, in a cluttered London laboratory by a man with a curved spine and a bottomless grudge against Isaac Newton.

## 1.5 Why the Law Had to Come First

There is a lesson in Galileo's partial success that runs through the entire history of engineering science. Mathematical frameworks can be erected before their foundations are complete, and they often are — because the problems are pressing and the people are clever. But structures built on incomplete foundations eventually require reconstruction. The history of beam theory is a long process of reconstruction: Galileo's formula, then Mariotte's correction, then the full Euler-Bernoulli theory, then the refinements of Timoshenko for thick beams, then the computational methods of finite element analysis. Each step added a piece that the previous step had approximated or guessed.

The piece that triggered the whole sequence of improvements was a simple observation about springs: that the force required to stretch a spring is proportional to how much you stretch it. No more, no less. A proportionality so obvious, once you measure it, that it seems almost too simple to have been a discovery.

It was a discovery. It had a discoverer. And he went to considerable trouble to keep it secret.

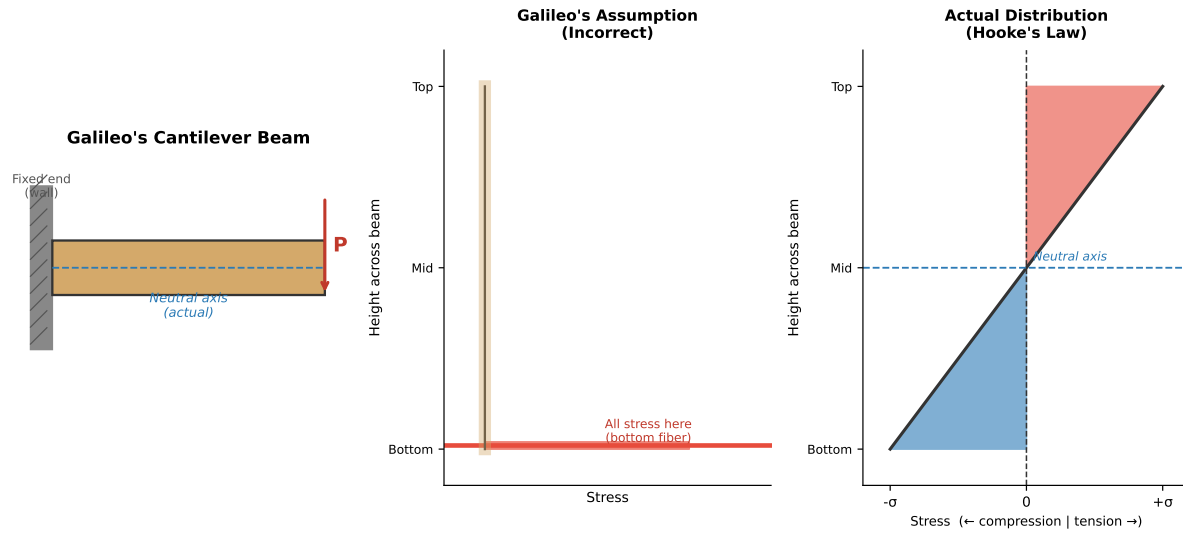


Figure 1.1: Galileo's cantilever beam problem. A horizontal beam fixed at the wall (left) carries a load  $P$  at its free end (right). The actual stress distribution across the cross-section at the wall is linear, varying from maximum tension at the top to maximum compression at the bottom, with zero stress at the neutral axis. Galileo incorrectly assumed all tensile resistance concentrated at the bottom fiber, leading to a factor-of-three overestimate of beam strength.

# Chapter 2

## Preface

Every object you have ever touched has pushed back.

This is not a trivial observation. It is, in fact, one of the most consequential facts in the history of engineering. When you press a table, the table presses you. When you stretch a rubber band, the rubber band pulls against your fingers. When a bridge carries a truck, the steel resists — not randomly, not arbitrarily, but in precise proportion to the load applied. Force in, force out, in a ratio that depends only on the material and the geometry and nothing else.

That proportionality has a name: Hooke's Law. It is stated in a handful of symbols. It was discovered, officially, in a handful of words hidden inside a Latin anagram in 1676. And it underpins virtually everything that engineers have built since.

This book is about that law — where it came from, how it grew, and how far it reaches. It is not a textbook. The equations are here, because the equations are part of the story and because understanding them, even imperfectly, is more satisfying than being told they exist. But the equations come after the people, not before. Robert Hooke comes before  $F = kx$ . Thomas Young comes before  $\sigma = E\varepsilon$ . Augustin-Louis Cauchy comes before the stress tensor. In each case, knowing the person makes the mathematics feel less like a formula and more like a discovery — which is what it was.

The law's modern reach is wider than most people realize. A seismologist uses it to interpret earthquake waves. An electrical engineer uses it to explain how a quartz crystal keeps time. A surgeon uses it, implicitly, every time she chooses a bone implant material. The accelerometer in your phone rests on a microscale spring whose behavior is governed by the same equation Hooke published when Newton was still a young man.

The chapters that follow trace this progression from its natural beginning: before the law existed. Galileo, who understood everything else, could not quite solve the beam problem because he lacked the piece that Hooke would supply a generation later. The history of structural mechanics is, to a remarkable degree, the history of one proportionality being understood more and more deeply — and applied further and further from the spring it started with.

A note on the equations. Every important equation in this book is explained in words before and after it appears in symbols. A reader who prefers to follow the narrative without pausing on the mathematics can do so; the prose carries the argument independently. A reader who wants

to work through the algebra will find the symbolic statements alongside their prose equivalents. Both readings are valid. The equations are not decoration, but they are not gates either.

*Troy Altus 2026*

## Chapter 3

# The Anagram and the Spring

### Learning Objectives

- Understand the life and context of Robert Hooke, and why his personal circumstances shaped his scientific methods
- Follow the discovery of Hooke's Law from experimental observation to published statement
- Interpret the law mathematically as  $F$  equals  $k$  times  $x$ , and understand what the spring constant  $k$  represents
- Recognize the Hooke-Newton rivalry as a defining conflict in the early Royal Society

### 3.1 A Difficult Man with a Difficult Life

Robert Hooke was, by nearly all accounts, not easy company. Contemporary descriptions portray a man lean to the point of emaciation, with a pronounced curvature of the spine — possibly scoliosis, possibly the result of childhood illness — and eyes that contemporaries described as grey, sharp, and probing. John Aubrey, who liked him, said his forehead was large and he had a quick, restless mind. Others were less charitable. He was suspicious of credit, litigious about priority, and carried grudges with the fidelity of a notary.

He was also, without serious competition, the most productive experimental scientist of the seventeenth century.

Hooke joined the Royal Society in 1660, shortly after its founding, and was appointed Curator of Experiments — meaning it was his job to design and conduct demonstrations at each meeting. The position was underpaid, occasionally humiliating, and required a weekly production of scientific novelty that would have broken most people. Hooke sustained it for decades. He invented or improved the balance spring for watches, the compound microscope (his *Micrographia* of 1665 is one of the great books in the history of science), the air pump, the wheel barometer, the iris diaphragm, and various other instruments whose names have been absorbed into the background of modern technology. He was the first person to observe plant cells under a microscope and to use the word “cell” to describe them.

None of this made him wealthy. Most of it made him resentful, because Robert Hooke's second

defining characteristic — after productivity — was his conviction that other people were taking credit for his ideas. In several notable cases, he was right.

## 3.2 The Longitude Problem

To understand why Hooke was working on springs in the 1660s and 1670s, you need to understand one of the great practical problems of the seventeenth century: longitude at sea.

Determining latitude — how far north or south you are — is relatively straightforward. Measure the angle of the sun at noon, or the North Star at night, and a table of values tells you where you stand. But longitude — how far east or west — requires knowing the exact time. The Earth rotates 360 degrees in 24 hours, which means it rotates 15 degrees per hour. If you know what time it is at a reference location (Greenwich, say) and you observe the local noon, the difference between those two times tells you your longitude directly. One hour of difference equals 15 degrees of longitude.

The problem was timekeeping. A clock that loses a few minutes per day on land is merely inconvenient. A clock that loses the same few minutes per day at sea will, over the course of a transatlantic voyage, accumulate enough error to place you a hundred miles from where you think you are. This was not a theoretical problem. Ships were regularly lost because navigators did not know their position.

Hooke believed that a watch regulated by a coiled spring — rather than by a pendulum, which is sensitive to the motion of a ship — could keep accurate enough time for longitude determination. This belief drove a great deal of his experimental work on springs. He was trying to understand how springs vibrate, how they age, how their restoring force behaves under repeated cycling. In the course of this practical investigation, he made the discovery that would outlast all of his other contributions.

## 3.3 The Anagram

In 1676, Hooke published a Latin pamphlet on helioscopes — devices for observing the sun — and appended to it, almost as a footnote, the following sequence of letters:

*ceiinosssttu*

This was not a typographical error. It was an anagram, and it was Hooke's deliberately obscure method of claiming priority for a discovery he was not yet ready to reveal. The convention was common enough in the seventeenth century: publish a scrambled version of your result so that, if someone else later announced the same discovery, you could unscramble your anagram and prove you had known it first. It was priority insurance with a secrecy premium.

The letters, rearranged, spell: *ut tensio sic vis*.

This is Latin for *as the extension, so the force*. Two years later, in 1678, Hooke published *Lectures de Potentia Restitutiva*, or *Of Spring*, which revealed the anagram and elaborated the law it encoded. The law, in his words: “The power of any spring is in the same proportion with the tension thereof.” Double the stretch, double the force. Halve the stretch, halve the force. The relationship is linear, and the constant of proportionality is a property of the spring.

### 3.4 The Experiment

Hooke's experimental setup was elegant in its simplicity. He hung a spring vertically, attached a pan to its lower end, and added weights to the pan one at a time. For each weight added, he measured how much the spring stretched. He then plotted — either explicitly or in his head, the historical record is not entirely clear — the force against the extension.

The result was a straight line through the origin. This is what “linear” means: a constant ratio between cause and effect. Twice the weight, twice the extension. Three times the weight, three times the extension. The line does not curve. It does not accelerate. It goes straight, at least until the spring is overloaded and its behavior changes fundamentally — a complication that Hooke noted but set aside.

In the notation we use today, Hooke's Law is written as:

$$F = kx \quad (3.1)$$

In this equation,  $F$  is the force applied to the spring, measured in Newtons. The letter  $x$  is the displacement of the spring from its natural, unloaded length — how much it has been stretched or compressed — measured in meters. And  $k$  is the spring constant, measured in Newtons per meter, a number that characterizes the stiffness of that particular spring.

The spring constant  $k$  carries all the information about the spring's resistance to deformation. A stiff spring — a heavy-duty automobile suspension spring — has a large  $k$ . A light spring — the kind that clicks your pen closed — has a small  $k$ . The law itself says nothing about what  $k$  is; that depends on the material the spring is made of, how thick the wire is, how many coils it has, and how tightly they are wound. Different springs, different  $k$  values. Same law.

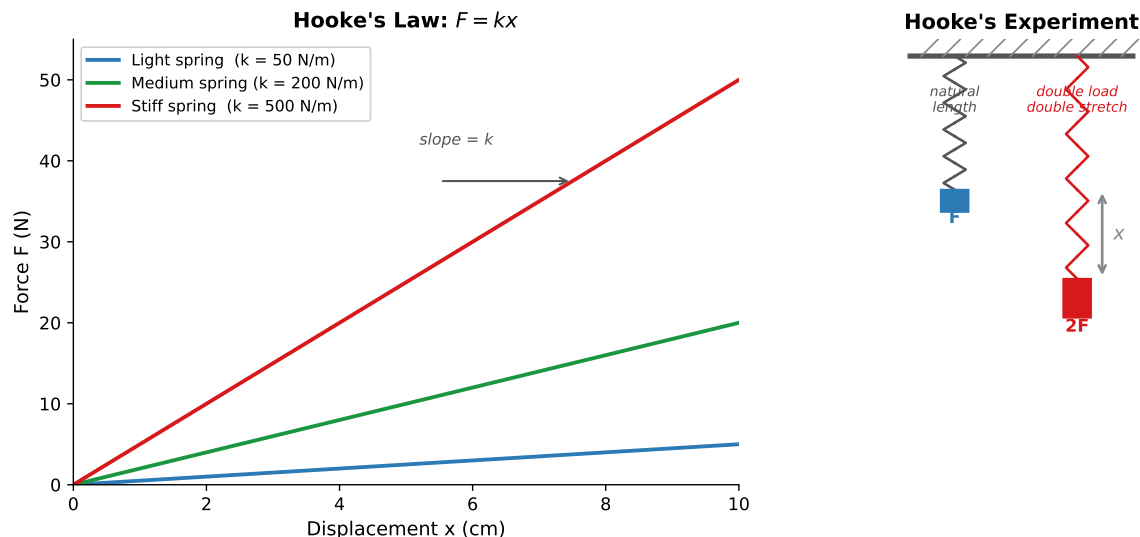


Figure 3.1: Hooke's Law illustrated for three springs of different stiffness. Force  $F$  is plotted against displacement  $x$  from the natural length. Each spring produces a straight line through the origin — the hallmark of linear proportionality. The slope of each line is the spring constant  $k$ , in units of Newtons per meter. A stiffer spring (larger  $k$ ) produces a steeper line: more force is required to achieve the same displacement.

### 3.5 What the Spring Constant Means

The spring constant  $k$  is a number with a physical meaning: it tells you how many Newtons of force are required to stretch the spring by one meter. A spring with  $k = 100$  Newtons per meter requires 100 N to stretch it one meter, 50 N to stretch it half a meter, and 10 N to stretch it a tenth of a meter. The relationship is exact, not approximate, as long as the spring is not overloaded.

What determines  $k$ ? For a coil spring, the answer involves the material's resistance to twisting (what we now call the shear modulus), the wire diameter, the coil diameter, and the number of active coils. Change any of these, and you change  $k$ . The law  $F = kx$  encapsulates all of this complexity into a single constant. This is one of the great virtues of Hooke's formulation: it is exact for any spring regardless of its geometry, as long as you know its  $k$ . The physics of *why*  $k$  has a particular value is a separate and more detailed question. The law itself says only that, whatever  $k$  is, the force and extension are proportional.

This separation — between the law's structure and the specific material constants that feed into it — is characteristic of the best physical laws. Newton's second law,  $F = ma$ , has the same quality: the law itself says force is proportional to acceleration; the mass  $m$  is a property of the object that you measure separately. The structure and the parameters are cleanly divided.

### 3.6 The Newton Problem

Robert Hooke's relationship with Isaac Newton was, to put it diplomatically, charged.

The conflict had many roots, but the most famous involved the inverse-square law of gravity. Hooke had, in 1679, written to Newton suggesting that orbital dynamics might follow an inverse-square relationship — that the gravitational force between two bodies might decrease as the square of the distance between them. Newton did not reply promptly. When he published his *Principia Mathematica* in 1687, which derived the orbits of planets from precisely this inverse-square law, Hooke erupted. He accused Newton of plagiarism, of using his ideas without credit. Newton, who had his own complex relationship with credit and with virtually everyone he worked with, denied any debt to Hooke and responded to the accusation with the implacable fury that he brought to most disputes.

The disagreement was never resolved. When Hooke died in 1703, Newton became President of the Royal Society. Shortly afterward, the only known portrait of Hooke — painted during his lifetime, the only image that might have shown us his face — disappeared. Whether Newton arranged its removal, or whether it simply vanished in the chaos of institutional transition, has never been established. What is established is that no authenticated portrait of Robert Hooke exists today.

For a man so concerned with priority and credit, this is a bitter irony. His name is on the law that underpins structural mechanics, seismology, watchmaking, and materials science. His face is unknown.

### 3.7 The Reach of a Simple Proportionality

In 1678, Hooke's Law was a useful description of spring behavior and not much more. Hooke himself used it primarily to analyze the behavior of balance springs in watches — a practical

engineering application, valuable but narrow. He understood that the law probably applied to other materials as well; he suggested as much in *Lectures de Potentia Restitutiva*, noting that the elastic properties of springs, arches, and bowed rods all followed the same proportion. But the systematic extension of the law from springs to solid materials — from  $F = kx$  to the relationship between stress and strain that governs the behavior of steel, concrete, glass, and bone — was the work of the following century, and it belongs principally to a man who read Hooke, understood him, and went considerably further.

That man could decode Egyptian hieroglyphics. He understood the wave theory of light well before it was accepted. He was a physician, a physicist, and a linguist. His name is not as famous as it should be, which is a persistent hazard in the history of science, and which we will address in the next chapter.



## Chapter 4

# From Spring to Stuff

### Learning Objectives

- Understand the conceptual problem with the spring constant  $k$  as a material descriptor
- Follow Thomas Young's reasoning that led to the modulus of elasticity
- Define stress as force divided by cross-sectional area, and strain as extension divided by original length
- Apply Young's Law in the form  $\sigma = E \epsilon$ , and interpret  $E$  as a material property independent of geometry
- Connect Young's modulus back to the spring constant through the relationship  $k = EA/L$

### 4.1 The Problem with Springs

Hooke's Law, in its original form, is a statement about springs. A spring has a spring constant,  $k$ , and the force required to stretch it by a distance  $x$  is  $F = kx$ . The law is precise, experimentally robust, and genuinely useful for analyzing anything that behaves like a spring.

But springs are peculiar objects. Their stiffness depends not only on the material they are made from but on their geometry: the wire diameter, the coil diameter, the number of turns. Two springs made from exactly the same grade of steel — the same atoms, arranged in the same crystal structure, with the same composition and heat treatment — can have spring constants that differ by a factor of a hundred, simply because one is wound tightly and the other loosely. This means that  $k$ , as Hooke defined it, is not a property of the steel. It is a property of the spring.

This is a problem if you want to compare materials. If you are trying to decide whether to build a bridge from iron or from oak — a genuinely practical question in the eighteenth century — you cannot compare their spring constants, because the spring constants depend on what shape you have made the material into. You need a property that belongs to the material itself, stripped of the geometry of the object. You need, in the vocabulary that would eventually emerge, a *material property* rather than a *structural property*.

The person who saw this distinction clearly, who named the required property and gave it a

definition rigorous enough to be used in calculation, was Thomas Young, in a series of lectures delivered at the Royal Institution in London beginning in 1801 and published in 1807.

## 4.2 Thomas Young, Polymath

Thomas Young was the sort of person who makes everyone else feel inadequate. He had learned to read by the age of two, and had read through the Bible twice by the time he was four. At the age of fourteen he was studying ten languages; he eventually became fluent in a dozen, including classical Greek, Hebrew, Persian, and Arabic. As a young physician in London, he wrote papers on the theory of the eye and color perception — laying groundwork for what we now call the trichromatic theory of color vision — while maintaining a medical practice. He made significant contributions to the understanding of light's wave nature, conducting the double-slit interference experiment that remains a fixture of every introductory physics course. And when the Rosetta Stone arrived in London in 1802, it was Young who made the first significant progress in deciphering Egyptian hieroglyphics, correctly identifying several of the phonetic characters.

In the interstices of all this, he gave lectures at the Royal Institution on natural philosophy and delivered, almost in passing, the definition of what we now call Young's Modulus.

Young was not a man of elegant self-promotion. His scientific papers were frequently too compressed for other readers to follow, and his lectures, according to contemporaries, were delivered at a speed and density that left audiences behind. He was brilliant in a way that outran communication — a recurring problem in the history of science, and one that has cost more than a few discoverers the credit they deserved. Young's contribution to elasticity theory was largely absorbed and clarified by French mathematicians over the following decade, and the modulus that bears his name was established in textbooks through the work of others as much as his own.

None of which diminishes the insight. The insight was real, it was important, and it was Young's.

## 4.3 Normalizing Away the Geometry

Young's key move was to recognize that if you want to compare materials, you need to normalize out the effects of size.

Consider two rods made from the same steel. One is long and thin; one is short and fat. If you apply the same force to each, the long thin rod stretches more than the short fat one. This is not because the material is different. It is because the geometry is different. A longer rod has more material to contribute to the elongation; a thinner rod has less cross-sectional area to distribute the load across.

To make a fair comparison, Young argued, you need to express the load in terms of force per unit area — how much force is carried by each square meter of cross-section — and you need to express the deformation in terms of elongation per unit original length — what fraction of its original length the material has stretched. These two quantities are independent of the size and shape of the sample. They are properties of the *material's response* to loading, not properties of the particular rod or bar being tested.

The first of these quantities is now called *stress*, denoted by the Greek letter sigma,  $\sigma$ . For a rod of cross-sectional area  $A$  carrying a load  $F$  in tension, the stress is:

$$\sigma = \frac{F}{A} \quad (4.1)$$

Stress is force divided by area, measured in Pascals, where one Pascal equals one Newton per square meter. When you press your thumb against a table, the table exerts a stress on your thumb that equals the force you are pushing with divided by the area of contact. A sharp thumbtack and a blunt thumb can exert the same total force; the tack concentrates that force onto a tiny area, producing a far higher stress — which is why the tack can penetrate the table and the thumb cannot.

The second quantity is called *strain*, denoted by the Greek letter epsilon,  $\varepsilon$ . For a rod of original length  $L$  that has stretched by an amount  $\Delta L$  under load, the strain is:

$$\varepsilon = \frac{\Delta L}{L} \quad (4.2)$$

Strain is dimensionless — it is a ratio of two lengths, so its units cancel. A steel rod that stretches by one millimeter from an original length of one meter has a strain of 0.001. A rubber band that stretches by ten centimeters from an original length of ten centimeters has a strain of 1.0. The numbers are directly comparable because they have been normalized to the original length.

Now Hooke's Law can be restated in terms of these normalized quantities. If force is proportional to extension — Hooke's original statement — then force per unit area is proportional to extension per unit length. In symbols:

$$\sigma = E\varepsilon \quad (4.3)$$

This equation is sometimes called Young's Law or, in older literature, the law of elasticity. The constant of proportionality  $E$  is called the *modulus of elasticity* or, most commonly, *Young's Modulus*. It is measured in Pascals, the same units as stress, because strain is dimensionless. Its value depends only on the material, not on the geometry of the sample.

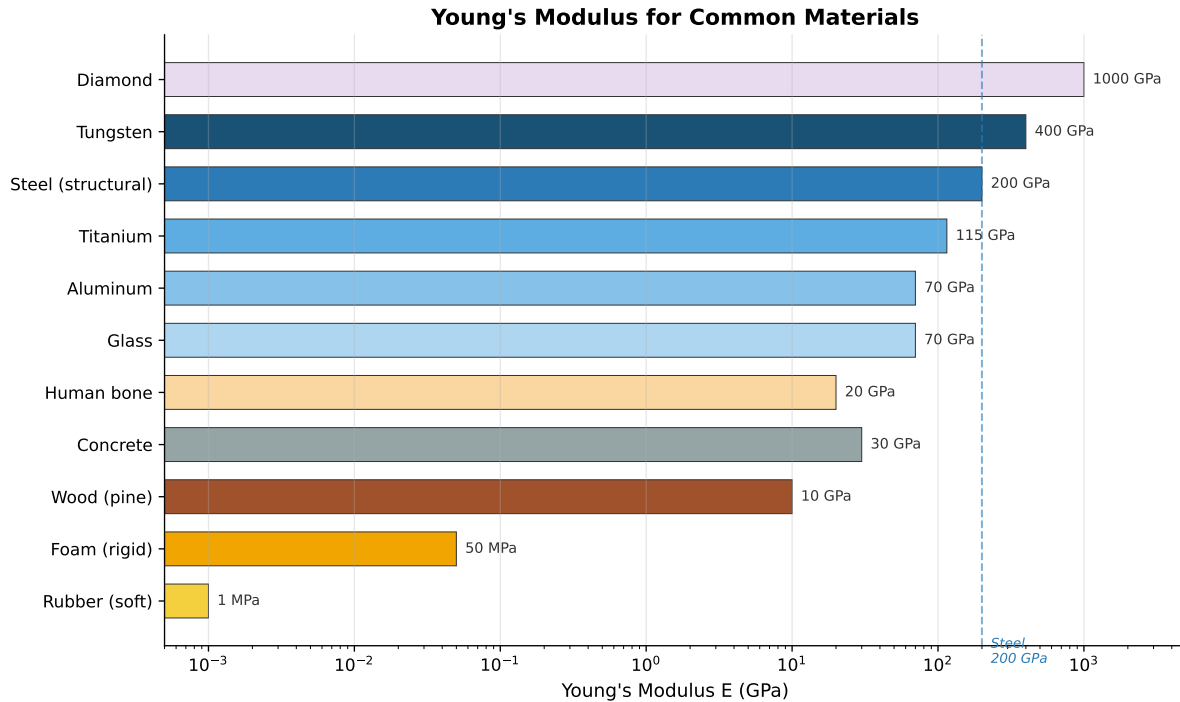


Figure 4.1: Young's Modulus for a selection of common engineering materials, plotted on a logarithmic scale in units of gigapascals (GPa, where one GPa equals one billion Pascals). The modulus spans more than four orders of magnitude across the materials shown — from rubber, which deforms under very small stresses, to diamond, the stiffest material known. For reference, structural steel has  $E$  approximately 200 GPa and human bone has  $E$  approximately 20 GPa. The logarithmic scale is necessary because the range of values is too large to display linearly.

## 4.4 Reading the Numbers

The chart above conveys, in a single image, the range of elastic behavior that engineers must navigate. Rubber, at one end of the scale, has a Young's Modulus near one megapascal — it takes only about one Newton applied over a square meter to produce a one-percent strain. Diamond, at the other end, has a modulus around a thousand gigapascals — one terapascal — and is so stiff that the strains produced by any practically achievable load are nearly unmeasurable.

Structural steel, at around 200 gigapascals, sits in a regime that engineers have been exploiting for a century and a half. This value means that applying a stress of 200 megapascals — about 200 million Newtons per square meter, a very high load for most applications — produces a strain of exactly one tenth of one percent. The steel has stretched by one millimeter per meter of original length. At this level of loading, the steel is still entirely elastic: release the load, and it springs back exactly to its original length. Hooke's Law holds precisely.

Human cortical bone, the dense outer shell of your femur and tibia, has a modulus around 20 gigapascals in the longitudinal direction. This is one-tenth the stiffness of steel, which means bone deforms ten times more under the same stress — but it is still far stiffer than most people intuitively expect. Bone is not soft. It is a remarkable composite material, and the reason your skeleton does not deflect visibly under your body weight is that its Young's Modulus is high

enough to produce strains that are mechanically significant but geometrically imperceptible.

## 4.5 The Spring Constant Revisited

Young's formulation connects back to Hooke's original spring constant in a satisfying way. For a rod of cross-sectional area  $A$ , original length  $L$ , and material modulus  $E$ , the spring constant  $k$  that relates the total applied force  $F$  to the total elongation  $\Delta L$  is:

$$k = \frac{EA}{L} \quad (4.4)$$

This equation says that a rod is stiffer — has a higher spring constant — if it is made from a stiffer material (larger  $E$ ), if it has a larger cross-section (larger  $A$ ), or if it is shorter (smaller  $L$ ). All three of these effects are intuitively correct and have been known empirically since antiquity. The equation quantifies them precisely and shows that they are all manifestations of the same underlying law.

A coil spring is more complicated than a straight rod — the wire is loaded in torsion rather than tension as the spring compresses or extends — but the same logic applies: the spring constant of a coil spring can be derived from the shear modulus of the wire material, the wire geometry, and the coil geometry. The material property (this time the shear modulus rather than Young's Modulus) is at the foundation; the spring constant emerges from combining that material property with the geometry.

## 4.6 Why This Matters

Young's reformulation of Hooke's Law shifted the language of engineering from objects to materials. Before Young, engineers compared springs. After Young, engineers could compare steels, and then compare steel to cast iron, to wrought iron, to bronze, to wood, to concrete. The modulus was something you could look up in a table, or measure in a laboratory on a small sample, and then use to predict the behavior of a large structure made from the same material.

This was the intellectual prerequisite for the explosion of structural engineering that followed the industrial revolution. The great iron bridges of the nineteenth century — Telford's Menai Suspension Bridge, Brunel's Clifton Suspension Bridge, the railway viaducts of the Victorian era — were all designed using methods that depended, at their foundation, on knowing the Young's Modulus of the iron in question. The modulus was measured; the allowable stress was specified; the required cross-sections were calculated. This is modern structural engineering, and it traces directly to Young's 1807 reformulation of what Hooke had observed in 1678.

But Young's formulation, powerful as it was, addressed only one dimension. A rod pulled in tension is a clean one-dimensional problem: one stress, one strain, one modulus. Real structures experience loads in multiple directions simultaneously. A column carries compression while also bending. A vessel wall carries tension in two perpendicular directions at once. The full mathematical description of how a material responds to three-dimensional loading — the extension of Hooke's Law into the full richness of the solid material world — was accomplished by a French mathematician in the 1820s, and it required the invention of a new mathematical object to express what the material was actually doing.



# Chapter 5

## Cauchy's Gift

### Learning Objectives

- Understand why a single number cannot fully describe the state of stress at a point inside a loaded solid
- Identify the six independent components of the stress tensor at a point in three-dimensional space
- Define Poisson's ratio and explain why lateral contraction accompanies axial tension
- State the generalized Hooke's Law for an isotropic elastic solid in terms of  $E$  and  $\nu$
- Recognize that two material constants are sufficient to fully describe linear elastic behavior for isotropic materials

### 5.1 The Trouble with One Dimension

A steel rod pulled in tension is the most convenient object in structural mechanics. It has one load direction, one stress, one strain, and one material constant. The problem fits on a single line of algebra. It is, in a meaningful sense, not the world.

Real structures are three-dimensional. A bolt in a flange carries tension along its axis and shear across its threads, simultaneously. A pressure vessel wall is stretched in two perpendicular directions at once — circumferentially and longitudinally — and compressed through its thickness. The corner of a steel frame, where a beam meets a column, carries combinations of bending moment, axial load, and shear that interact in ways a one-dimensional analysis cannot capture.

What Thomas Young gave engineers was essential but incomplete. He provided a material property — the modulus of elasticity — and a one-dimensional relationship between stress and strain. What he could not provide, because the mathematical tools did not yet exist in convenient form, was a description of stress and strain as three-dimensional quantities. He knew that the one-dimensional version was incomplete. The completion was the work of Augustin-Louis Cauchy, working in Paris in the 1820s.

## 5.2 Cauchy

Augustin-Louis Cauchy was among the most productive mathematicians in history, which is saying something in a field that produced Euler and Gauss in the same era. He published approximately 789 papers — a number that strains credulity until you read the papers and realize that many of them are short, dense communications on specific problems, the nineteenth-century equivalent of journal letters. He made fundamental contributions to real analysis (the rigorous foundation of calculus), complex analysis, differential equations, optics, and mechanics. Several theorems, a theorem in number theory, a formula in complex analysis, and two integral theorems carry his name.

He was also, by several accounts, insufferably self-righteous — a devout Catholic who occasionally refused to collaborate with scientists whose politics or religion he disapproved of, and who once declined a professorship rather than take an oath that conflicted with his religious convictions. This made him difficult in person and extraordinary on paper.

In 1822, Cauchy presented to the French Academy of Sciences a paper that established the modern concept of stress as a mathematical object. It was the missing piece that would allow Hooke's Law to be extended from the one-dimensional rod to the full three-dimensional solid.

## 5.3 Stress at a Point

Before Cauchy, engineers and mathematicians had a practical notion of stress — force per unit area — but they had not examined it carefully as a function of orientation. If you take a small element of material inside a loaded solid and ask what forces act on it, the answer depends on which face of the element you are looking at. A face oriented perpendicular to the loading direction experiences a different force per unit area than a face oriented parallel to it.

Cauchy's insight was to make this orientation-dependence precise and systematic. He defined the *traction* on a surface as the force per unit area acting on that surface, and he showed that the traction on any surface through a point can be calculated from the tractions on three mutually perpendicular surfaces through the same point. This means that the state of stress at a point — the complete description of all the forces acting within the material at that location — is captured by the tractions on three perpendicular faces.

Each of these three faces has a traction vector that can be decomposed into three components: one normal to the face (pushing or pulling it) and two parallel to the face (sliding it in two perpendicular directions). This gives three faces times three components each, for a total of nine numbers that completely describe the stress state.

These nine numbers form what is now called the *stress tensor*. In the compact notation of modern mechanics, it is written as a three-by-three matrix:

$$\sigma = \begin{pmatrix} \sigma_{xx} & \sigma_{xy} & \sigma_{xz} \\ \sigma_{yx} & \sigma_{yy} & \sigma_{yz} \\ \sigma_{zx} & \sigma_{zy} & \sigma_{zz} \end{pmatrix} \quad (5.1)$$

The diagonal components —  $\sigma_{xx}$ ,  $\sigma_{yy}$ , and  $\sigma_{zz}$  — are the *normal stresses*: they act perpendicular to their respective faces, pulling or pushing the material apart or together. The off-diagonal

components —  $\sigma_{xy}$ ,  $\sigma_{xz}$ ,  $\sigma_{yz}$ , and their counterparts — are the *shear stresses*: they act parallel to the faces, trying to slide one part of the material past another.

Cauchy also showed that the stress tensor must be symmetric — that is,  $\sigma_{xy} = \sigma_{yx}$ ,  $\sigma_{xz} = \sigma_{zx}$ , and  $\sigma_{yz} = \sigma_{zy}$ . This symmetry follows from the requirement that a small element of material not spin spontaneously under load — a condition called moment equilibrium. The symmetry reduces the nine independent numbers to six. Six numbers are required to fully describe the state of stress at a single point in a three-dimensional solid under load. No more, no less.

The strain tensor has exactly the same structure. Six independent components describe the deformation state at a point: three normal strains (elongation or contraction in three directions) and three shear strains (angular distortion in three planes).

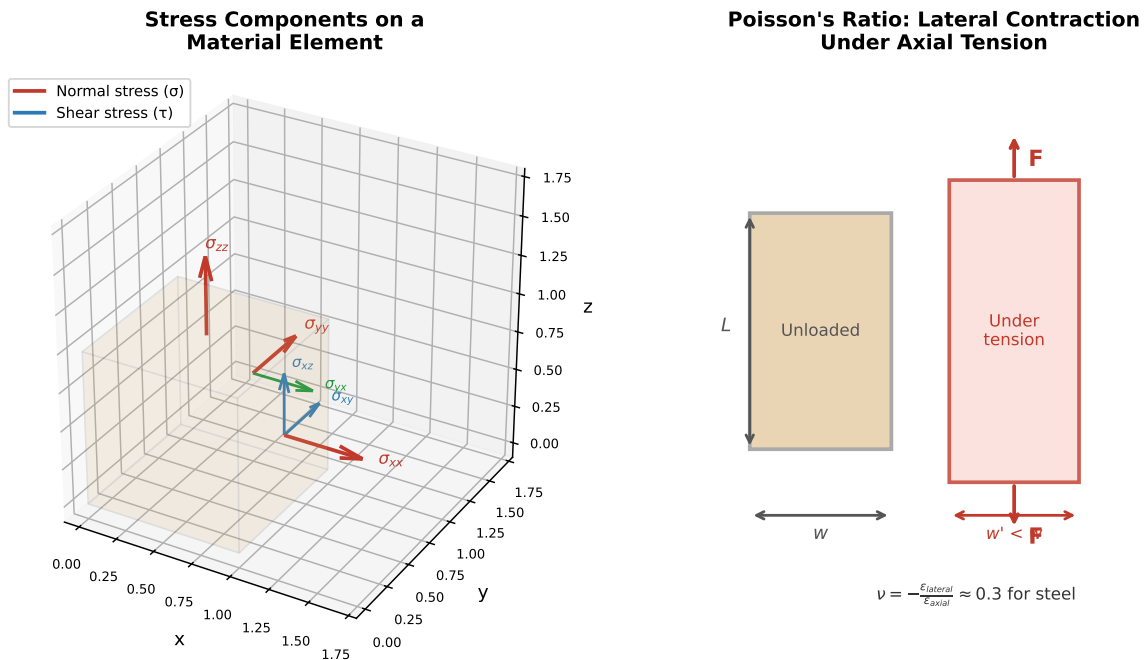


Figure 5.1: Stress components acting on a small cubic element of material. The three normal stresses —  $\sigma_{xx}$ ,  $\sigma_{yy}$ , and  $\sigma_{zz}$  — act perpendicular to the faces of the cube, in tension (pulling outward, shown) or compression (pushing inward). The six shear stresses —  $\sigma_{xy}$ ,  $\sigma_{xz}$ ,  $\sigma_{yx}$ ,  $\sigma_{yz}$ ,  $\sigma_{zx}$ ,  $\sigma_{zy}$  — act parallel to the faces, attempting to slide adjacent layers of material past one another. Moment equilibrium requires  $\sigma_{xy}$  to equal  $\sigma_{yx}$ ,  $\sigma_{xz}$  to equal  $\sigma_{zx}$ , and  $\sigma_{yz}$  to equal  $\sigma_{zy}$ , reducing the nine tensor components to six independent values.

## 5.4 Poisson's Ratio

While Cauchy was developing the tensor framework, his French colleague Siméon Denis Poisson was measuring something that every person has observed but few have quantified: when you stretch a material in one direction, it contracts in the perpendicular directions.

Pull a rubber band between your fingers. As it stretches in the direction you are pulling, it gets noticeably narrower. This lateral contraction is not an accident of rubber's construction. It is

a nearly universal property of elastic materials. Steel does it. Concrete does it. Bone does it. Even rock does it, though far less dramatically than rubber.

Poisson defined the ratio of lateral contraction to axial extension and called it, in effect, a material constant. We now call it *Poisson's ratio*, denoted by the Greek letter  $\nu$  (nu). For a rod stretched by an axial strain of  $\varepsilon_{axial}$ , the lateral strain is:

$$\varepsilon_{lateral} = -\nu\varepsilon_{axial} \quad (5.2)$$

The negative sign captures the fact that the lateral strain is compressive (negative) when the axial strain is tensile (positive). Poisson's ratio is a positive number for almost all common materials. For steel and aluminum,  $\nu$  is approximately 0.3. For rubber, it approaches 0.5 — meaning rubber is nearly incompressible under tension, it stretches without losing volume. For cork,  $\nu$  is close to zero, which is why corks seal wine bottles effectively: they compress in the axial direction without expanding laterally into the bottle neck.

## 5.5 The Generalized Hooke's Law

With Cauchy's stress tensor, the strain tensor, and Poisson's ratio in hand, the three-dimensional generalization of Hooke's Law can be stated. For an isotropic material — one whose elastic properties are the same in all directions — the relationship between stress and strain requires exactly two material constants: Young's Modulus  $E$  and Poisson's ratio  $\nu$ .

The three normal strain components are related to the three normal stress components by:

$$\varepsilon_{xx} = \frac{1}{E} [\sigma_{xx} - \nu(\sigma_{yy} + \sigma_{zz})] \quad (5.3)$$

$$\varepsilon_{yy} = \frac{1}{E} [\sigma_{yy} - \nu(\sigma_{xx} + \sigma_{zz})] \quad (5.4)$$

$$\varepsilon_{zz} = \frac{1}{E} [\sigma_{zz} - \nu(\sigma_{xx} + \sigma_{yy})] \quad (5.5)$$

Reading these equations in words: the strain in the  $x$ -direction equals the axial stress in  $x$  divided by Young's Modulus, minus Poisson's ratio times the sum of the stresses in the other two directions divided by Young's Modulus. Stress in one direction produces strain in all three directions. This is the coupling that Hooke's one-dimensional law could not express: every stress component affects every strain component, and the coupling is governed by Poisson's ratio.

The shear strains are related to the shear stresses through the shear modulus  $G$ :

$$\varepsilon_{xy} = \frac{\sigma_{xy}}{2G}, \quad \varepsilon_{xz} = \frac{\sigma_{xz}}{2G}, \quad \varepsilon_{yz} = \frac{\sigma_{yz}}{2G} \quad (5.6)$$

Importantly,  $G$  is not a third independent material constant. For an isotropic material, it is determined by  $E$  and  $\nu$ :

$$G = \frac{E}{2(1 + \nu)} \quad (5.7)$$

So two numbers —  $E$  and  $\nu$  — completely specify the linear elastic behavior of an isotropic material. Every structural calculation involving steel, aluminum, glass, or concrete ultimately rests on these two numbers and the tensor equations that Cauchy and Poisson provided.

## 5.6 The Achievement

Stepping back, the achievement is remarkable. In roughly 150 years — from Hooke’s 1678 anagram to Cauchy’s 1822 paper — the science of elasticity went from a qualitative observation about spring behavior to a complete mathematical framework for three-dimensional deformation of solid bodies. The key steps were: the discovery of linear proportionality between force and displacement (Hooke); the reformulation in terms of normalized, geometry-independent quantities (Young); and the extension to full three-dimensional tensor form (Cauchy and Poisson).

Each step was driven by practical need. Engineers were building larger bridges, larger ships, larger machines. They needed to predict, not merely guess. The mathematical framework that emerged from this need is, in its essentials, what every structural engineer uses today — though the calculations are now performed by computers and the structures involved would have been unimaginable to Hooke.

What remained, in 1822, was the application. Knowing how a material element responds to stress is not the same as knowing how a beam deflects, or how a column buckles, or how a pressure vessel fails. Translating material behavior into structural behavior requires assembling many infinitesimal elements — each obeying Hooke’s Law — into a complete structure, and determining how that structure deforms as a whole. This is the business of structural mechanics, and it has its own elegant results, which we take up in the next chapter.



# Chapter 6

## The Bent Beam

### Learning Objectives

- Derive the Euler-Bernoulli bending equation relating curvature, moment, and the material and geometric properties of a beam
- Understand the second moment of area and why it governs bending stiffness
- Explain why the I-beam cross-section is the efficient structural form for bending loads
- Apply Euler's critical load formula for column buckling and interpret the slenderness ratio
- Return to Galileo's cantilever error and show, using Hooke's Law, why Mariotte's neutral axis is correct

### 6.1 The Beam Reconsidered

Galileo's cantilever problem, which we visited in the first chapter, sat unresolved for most of the seventeenth century. The missing ingredient was Hooke's Law. With it — with the knowledge that stress is proportional to strain, and that strain in a bent beam varies linearly with distance from the neutral axis — the problem becomes tractable. Without it, the stress distribution across the cross-section is a guess, and Galileo's guess was wrong.

By the late eighteenth century, the tools were in place. Young's Modulus existed as a material property. The geometry of bending had been studied by Jacob Bernoulli and, more rigorously, by Euler. What emerged from their combined effort is now called Euler-Bernoulli beam theory, and it remains the foundation of structural design for beams, girders, joists, and virtually every other horizontal load-carrying member in common use.

### 6.2 How a Beam Bends

Imagine a simply supported beam — horizontal, resting on supports at both ends, carrying a load somewhere in the middle. Under the load, the beam sags. The top surface is compressed; the material there has gotten shorter. The bottom surface is stretched; the material there has gotten longer. Somewhere between top and bottom, the material has changed length by exactly zero — it is neither compressed nor stretched. This is the neutral axis.

Hooke's Law requires that the stress at any point in the cross-section be proportional to the strain at that point, which is proportional to the distance from the neutral axis. The stress distribution is therefore linear: zero at the neutral axis, maximum at the outer fibers, tensile below and compressive above (for a downward load). This is precisely what Mariotte guessed and what Galileo missed. The mathematical proof that this is the *only* consistent distribution for a material obeying Hooke's Law came later, but the physical intuition was correct.

The bending moment  $M$  at a cross-section — the net turning effect of all the external forces to one side of that section — must be balanced by the internal stress distribution. Working through the geometry and applying Hooke's Law, one arrives at the fundamental relationship:

$$M = EI\kappa \tag{6.1}$$

In this equation,  $E$  is Young's Modulus,  $I$  is the second moment of area of the cross-section about the neutral axis, and  $\kappa$  is the curvature of the beam at that point — how sharply the beam is bent. For small deflections, the curvature is approximately equal to the second derivative of the deflection  $y$  with respect to the axial position  $x$ , giving the differential equation that governs beam deflection:

$$EI \frac{d^2y}{dx^2} = M(x) \tag{6.2}$$

This is the Euler-Bernoulli beam equation, and it contains Hooke's Law in its DNA. The  $E$  is Young's Modulus; the  $I$  is geometry; the product  $EI$  is called the *flexural rigidity* of the beam. Double the modulus and the beam deflects half as much. Double the second moment of area and the beam deflects half as much. The two effects are interchangeable, which is a significant design insight.

### 6.3 The Second Moment of Area

The second moment of area,  $I$ , is worth examining carefully, because it explains one of the most consequential design decisions in structural engineering: the shape of the cross-section.

For a rectangular cross-section of width  $b$  and height  $h$ , the second moment of area about the horizontal neutral axis is:

$$I = \frac{bh^3}{12} \tag{6.3}$$

The cubic dependence on height is striking. Double the height of a rectangular beam and you multiply its bending stiffness by eight — not two, not four, but eight. A beam twice as tall is eight times stiffer in bending. This is why tall beams carry bending loads more efficiently than wide, flat ones.

But there is a subtlety. The second moment of area rewards material that is *far from the neutral axis*. Material near the neutral axis contributes little to bending resistance — its stress and strain are both small. Material at the outer fibers contributes the most. A solid rectangular section carries material at all distances from the neutral axis, including the low-value region

near the center. A hollow section, or an I-shaped section, removes material from near the center and concentrates it at the flanges, where it does the most good.

This is the engineering logic of the I-beam. By concentrating material in the flanges — far from the neutral axis — and using a thin web to connect them and carry shear, the I-beam achieves a high second moment of area with a relatively small cross-sectional area, meaning high bending stiffness with low material consumption. The I-beam is not an aesthetic preference or a manufacturing accident. It is the direct structural consequence of Hooke's Law applied to bending.

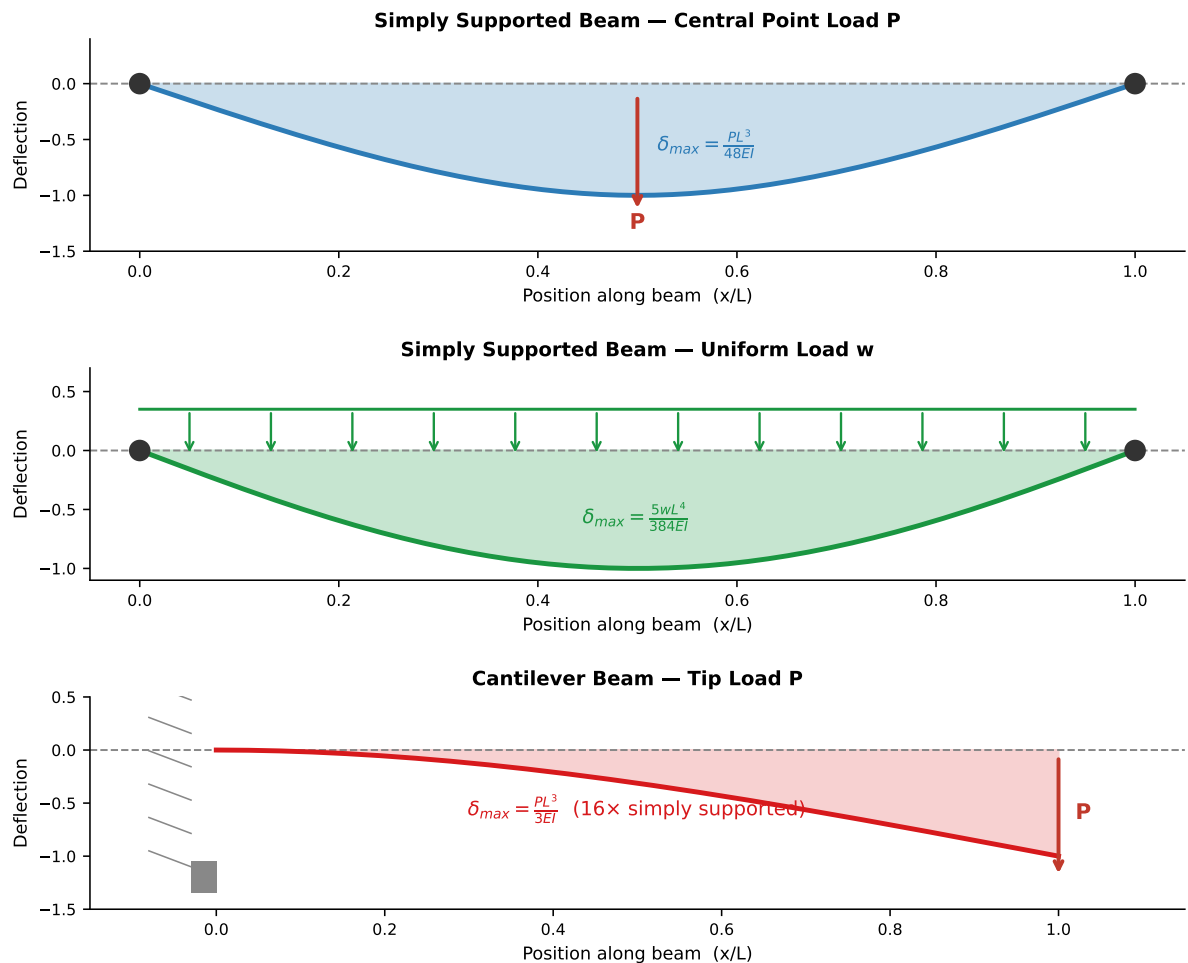


Figure 6.1: Beam deflection under three common loading conditions, calculated using the Euler-Bernoulli beam equation. All beams have the same flexural rigidity  $EI$  and the same span  $L$ . Top: a simply supported beam carrying a central point load  $P$  deflects by  $PL$ -cubed divided by  $48EI$  at the center. Middle: a uniformly distributed load  $w$  produces a maximum deflection of  $5wL$ -to-the-fourth divided by  $384EI$  at the center. Bottom: a cantilever beam with a tip load  $P$  deflects by  $PL$ -cubed divided by  $3EI$  at the free end — sixteen times more than a simply supported beam of the same span under the same load, because only one end is supported. Note that deflections are exaggerated for clarity; real structural beams deflect by much smaller amounts relative to their span.

## 6.4 Euler's Column

Beams carry load in bending. Columns carry load in compression. The behavior of slender columns under compressive load is governed by a relationship that Euler derived in 1744 — the same Euler who co-developed beam bending theory — and it reveals one of the more unsettling truths in structural mechanics: a column can fail not by being crushed but by suddenly springing sideways.

This phenomenon is called *buckling*, and it illustrates something subtle about Hooke's Law's role in structural behavior. The material inside a column under compressive load is behaving exactly as Hooke's Law predicts — stress proportional to strain, perfectly elastic, no permanent deformation. And yet the column fails. It fails not because the material yields, but because the straight configuration becomes geometrically unstable.

Euler showed that there is a critical load,  $P_{cr}$ , at which a perfectly straight, perfectly uniform column will buckle:

$$P_{cr} = \frac{\pi^2 EI}{(KL)^2} \quad (6.4)$$

In this equation,  $E$  is Young's Modulus,  $I$  is the second moment of area of the cross-section (the same one that governs bending),  $L$  is the length of the column, and  $K$  is an effective length factor that depends on how the column's ends are constrained. A column pinned at both ends has  $K = 1$ . A column fixed at both ends has  $K = 0.5$ , meaning it behaves like a column of half the length and carries four times the buckling load.

The factor  $(KL)^2$  in the denominator is telling. Buckling load decreases with the *square* of the effective length. Double the length and the critical load drops by a factor of four. This explains why slender columns fail at loads far below the material's compressive strength: the geometry defeats the material.

The ratio  $KL/r$  — where  $r$  is the radius of gyration,  $r = \sqrt{I/A}$ , a geometric property of the cross-section — is called the *slenderness ratio*. It is the single number that determines whether a column will buckle before it yields. High slenderness: buckling governs. Low slenderness: material crushing governs. The dividing line depends on the material's modulus and yield stress. For structural steel, columns with slenderness ratios above about 100 are firmly in the buckling-governed regime.

## 6.5 Correcting Galileo

With the Euler-Bernoulli beam equation in hand, we can finally resolve the error that Galileo made in 1638.

Galileo placed all the tensile resistance at the bottom fiber of the cantilever's cross-section, giving his formula for breaking load a fixed end moment equal to the force times the beam height. The correct analysis, using Hooke's Law and the section modulus  $S = I/c$  (where  $c$  is the distance from the neutral axis to the outer fiber), gives a breaking condition of:

$$\sigma_{max} = \frac{Mc}{I} = \frac{M}{S} \quad (6.5)$$

For a rectangular cross-section of width  $b$  and height  $h$ :  $I = bh^3/12$ ,  $c = h/2$ , and  $S = bh^2/6$ . Galileo's formula, by contrast, effectively used a section modulus of  $bh^2/2$  — three times larger. He overestimated the strength by a factor of three. Mariotte's correction, placing the neutral axis at the centroid, was qualitatively right but lacked the quantitative derivation. The full derivation required Hooke's Law, the concept of Young's Modulus, and the integration of stresses across the cross-section — tools that became available incrementally over the century following Galileo.

Galileo was not a sloppy thinker. He was solving a problem that required mathematical tools which did not exist in his lifetime. His error was, in a meaningful sense, not his fault. It was the fault of the missing law that a younger, querulous, brilliant man named Hooke would discover thirty years after Galileo's death.



# Chapter 7

## Where Hooke Lies

### Learning Objectives

- Interpret a complete engineering stress-strain curve from initial loading through fracture
- Define the proportionality limit, elastic limit, yield point, ultimate tensile strength, and fracture strain
- Understand plastic deformation in terms of dislocation motion in crystalline materials
- Apply Griffith's fracture mechanics insight that a crack amplifies stress locally and dramatically
- Recognize which materials obey Hooke's Law over useful ranges and which require different frameworks

### 7.1 A Material's Biography

When a metallurgist wants to understand a material, the first thing she does is pull it apart. Not carelessly — the pulling is controlled, the force is measured continuously, and the extension is recorded at every moment. The result is a curve of stress versus strain that begins with a straight line and ends, eventually, in fracture. This curve is a biography of the material's elastic and plastic life. Everything Hooke's Law can explain appears in the first part of the curve. Everything else appears after.

The stress-strain test is performed in a tensile testing machine: a device that grips a precisely machined sample at both ends and pulls them apart at a controlled rate, measuring the force required at every instant. The sample — typically a round bar or a flat coupon with a reduced-gauge section — is machined to standard dimensions so that the test can be compared to results from other laboratories on other machines. The force is divided by the original cross-sectional area to give engineering stress; the extension is divided by the original gauge length to give engineering strain. The resulting curve is a material property, not a property of the sample's dimensions.

## 7.2 The Five Territories

Steel — specifically a low-carbon structural steel, the workhorse of construction and the material that built the modern world — produces one of the more instructive stress-strain curves in engineering. It has five distinguishable regions, each governed by different physics.

**The elastic region** is where Hooke’s Law holds exactly. Stress is proportional to strain; the curve is a straight line with slope  $E$ , Young’s Modulus. If you release the load at any point in this region, the material returns to exactly its original dimensions. The deformation is completely reversible. The steel remembers its unloaded shape as if the loading had never happened. For structural steel, the elastic region extends to stresses of roughly 250 megapascals — about 250 million Newtons per square meter. This seems like a large number until you realize that the weight of a modest building, spread over the structural columns, can approach such stresses at the base.

**The proportionality limit** is the stress at which the curve first deviates from a perfectly straight line. Below this stress, the proportionality between stress and strain is exact. Above it, the curve begins — very slightly — to bend. The deviation is often too small to detect without precise instrumentation, and for practical engineering the proportionality limit is frequently taken as coincident with the elastic limit.

**The elastic limit** is the stress above which the material will no longer return to its original dimensions when unloaded. Load the material beyond this point and release; the sample will be slightly longer than it was. This permanent elongation is called *plastic strain*, and the material has now been *plastically deformed*. The change is permanent. The steel has, in a colloquial sense, learned a new shape.

**The yield point** is where the curve takes a dramatic turn in low-carbon structural steel — a turn so dramatic that it caused early materials testers to suspect instrument malfunction. Above the yield point, the stress actually *drops* and then oscillates at roughly constant value while the strain continues to increase. The material is deforming plastically at nearly constant stress. This plateau is called the *yield plateau* or *Lüders extension*, and it can extend to strains several times the elastic-limit strain before strain hardening begins.

**Strain hardening** is the gradual stiffening that occurs as the material continues to be stretched beyond the yield plateau. The curve rises again, though less steeply than in the elastic region, as the material’s crystal structure becomes increasingly disrupted and further deformation requires increasing stress.

**Ultimate tensile strength** is the peak of the curve: the maximum stress the material can carry. Beyond this point, the sample begins to *neck* — a local region of the gauge section contracts more rapidly than the rest, concentrating deformation in a small zone. The engineering stress (calculated using the original area) falls after necking begins, even though the true stress in the necked region is still rising.

**Fracture** is the end. The necked region reaches its limit and separates. The pieces can be reassembled and examined; the fracture surface records a great deal about the mode and mechanism of failure.

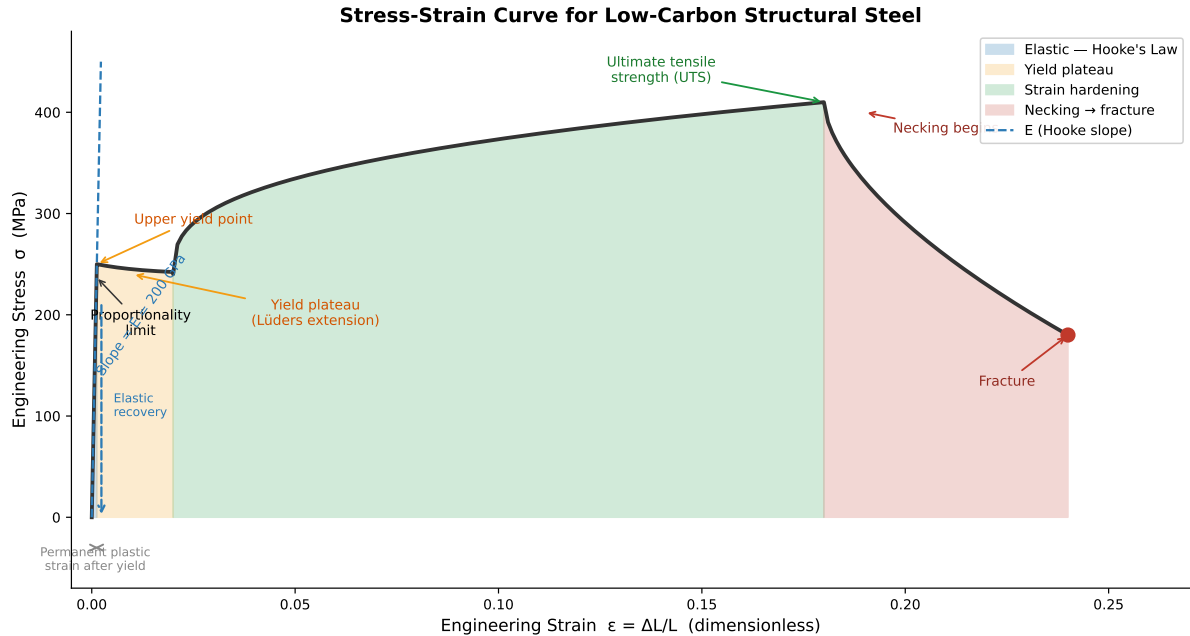


Figure 7.1: Engineering stress-strain curve for a low-carbon structural steel, showing the five principal regions of material behavior. The elastic region (blue) is governed by Hooke’s Law with slope equal to Young’s Modulus  $E$ . The yield point marks the transition from reversible elastic deformation to permanent plastic deformation. The yield plateau (orange) is characteristic of low-carbon steels; higher-strength alloys show a smooth transition without a distinct plateau. Strain hardening (green) continues until the ultimate tensile strength is reached, after which necking begins and the engineering stress falls until fracture. Note that the strain scale is compressed relative to the stress scale for clarity; the fracture strain of a ductile steel (approximately 0.20 to 0.25, or 20 to 25 percent elongation) is roughly 200 times the elastic-limit strain (approximately 0.001).

### 7.3 What Happens Inside

The stress-strain curve describes behavior from the outside — force and displacement, stress and strain. But the physics happens inside the material, at the level of atoms and crystal defects, and understanding that level explains why the curve has the shape it does.

Metals are crystalline: the atoms are arranged in regular, repeating lattices. In the elastic region, stretching the metal means stretching the bonds between atoms. These bonds behave, to a good approximation, like Hooke’s Law springs: they resist deformation linearly and spring back completely when released. Young’s Modulus is, at its deepest level, a measure of the stiffness of interatomic bonds.

When the yield stress is reached, something different happens. Rather than stretching the bonds until they break — which would require much higher stresses — the atoms slide past each other along specific crystallographic planes. The mechanism is not a wholesale sliding of one plane over another, which would require simultaneously breaking all the bonds across the interface. It is instead carried by *dislocations* — line defects in the crystal structure at which the lattice is misaligned by one atomic spacing. A dislocation moves through the crystal like a wrinkle

moving across a carpet: the wrinkle crosses the room with far less effort than dragging the whole carpet. Plastic deformation is dislocation motion, and the yield stress is the stress at which dislocations begin to move freely.

Strain hardening occurs because moving dislocations encounter other dislocations, grain boundaries, and precipitates, all of which impede further motion. As deformation continues, the dislocation density increases and the obstacles multiply, making further deformation progressively harder. The stress must rise to keep the dislocations moving. This is the rising portion of the curve after the yield plateau.

## 7.4 Griffith and the Crack

In 1920, an engineer at the Royal Aircraft Factory at Farnborough named A.A. Griffith set out to understand why glass fractures at stresses far below those that theoretical atomic bond strength would predict. Glass, calculated from the force required to separate two atomic planes, should be able to withstand stresses approaching ten gigapascals. In practice, glass fractures at stresses a hundredfold smaller. The discrepancy was enormous, and it was not explained by conventional strength-of-materials theory.

Griffith's answer was the crack. Real materials contain cracks — tiny, often invisible, sometimes just a few atoms wide. These cracks do not simply weaken the material by reducing the load-bearing area. They do something far more destructive: they concentrate stress at their tips.

Hooke's Law applied to the elastic field around a crack tip shows that the stress at the tip is theoretically infinite for a perfectly sharp crack — it diverges as  $1/\sqrt{r}$ , where  $r$  is the distance from the tip. In practice, real cracks are not perfectly sharp at the atomic scale, and real materials are not perfectly brittle — some plastic deformation occurs at the tip, blunting it and limiting the stress concentration. But the stress at a crack tip is vastly larger than the nominal applied stress, and this concentration is what causes brittle fracture at loads far below theoretical strength.

Griffith's formula for the stress required to propagate a crack of half-length  $a$  in a material with surface energy  $\gamma$  and modulus  $E$  is:

$$\sigma_f = \sqrt{\frac{2E\gamma}{\pi a}} \quad (7.1)$$

This equation predicts that fracture stress is inversely proportional to the square root of crack length. Double the crack length and you reduce the fracture stress by thirty percent. A crack one millimeter long reduces the fracture strength of glass by a factor of roughly thirty compared to its theoretical maximum. This is why glass is fragile: not because it is inherently weak, but because it is inherently cracked.

## 7.5 Where Hooke's Law Is and Is Not

The honest summary: Hooke's Law is correct for an enormous range of materials and conditions, and it is the foundation of structural design. But it has boundaries.

Metals obey Hooke's Law up to their yield point — which, for structural steel, is a practical and useful range. Beyond yield, they deform permanently.

Ceramics (glass, concrete, stone) obey Hooke's Law with high moduli but fracture with essentially no warning and no yielding — the stress-strain curve is a straight line that ends abruptly. Hooke's Law describes them perfectly right up to the moment they shatter.

Polymers and rubber do not, in general, obey Hooke's Law. Rubber can undergo strains of several hundred percent and still return to its original shape — but the force-extension curve is markedly nonlinear. Different frameworks are required, which we will touch on in the final chapter.

Biological tissues — tendon, muscle, skin, artery wall — are spectacularly nonlinear. A tendon stiffens dramatically as it is stretched: initially easy to extend, then progressively stiffer. This behavior, called *toe-region nonlinearity*, is caused by the progressive uncrimping of collagen fibers.

In all these cases, Hooke's Law fails to capture the full picture. But it often captures the initial, small-deformation behavior accurately, and the small-deformation regime is where most engineering structures spend most of their lives. The law's range of validity coincides, in most cases, with the regime of safe structural operation. To operate a structure in the regime where Hooke's Law fails is, almost always, to operate it in a regime where it is about to fail.

That coincidence is not a coincidence at all. It is the reason we care about Hooke's Law in the first place.



## Chapter 8

# Springs in Everything

### Learning Objectives

- Identify how Hooke's Law appears in seismology, piezoelectricity, finite element analysis, and biomechanics
- Understand why so many physical systems are linear near equilibrium through the perturbation argument
- Recognize the MEMS accelerometer as a modern realization of Hooke's original experiment at microscale
- Appreciate that the neo-Hookean model for rubber extends the linear law into finite deformation

## 8.1 A Law That Keeps Spreading

When Robert Hooke hung weights from a coiled spring in the 1670s, he was thinking about watch regulation. He wanted to build a better chronometer and, along the way, described the elastic behavior of springs with the precision of a law. It would have been reasonable, in 1678, to regard this as a specialized observation: useful for clockmakers, perhaps interesting to natural philosophers, but confined in its application to springs and similar elastic mechanisms.

Three and a half centuries later, Hooke's proportionality — in one form or another, extended and reinterpreted but never abandoned — appears in the mathematics of earthquake waves, in the crystal that regulates your wristwatch, in the software that designs aircraft and bridges, in the tiny silicon spring that tells your phone which way is up, and in the biological tissue that transmits force from your muscles to your bones. The law has proven to be not a description of springs but a description of elastic matter itself — and elastic matter, it turns out, is everywhere.

## 8.2 Seismology: The Earth Springs Back

When tectonic plates slip along a fault, the resulting earthquake releases energy that propagates through the Earth as elastic waves. These waves are elastic in the precise sense: the rock deforms as the wave passes, and it springs back. Hooke's Law governs the deformation. The wave equation for seismic waves is derived directly from Newton's second law applied to an

elastic solid, with the restoring force provided by the elastic stresses described by the generalized Hooke's Law.

Two types of elastic waves are relevant. *Primary waves* (P-waves) are compressional: the rock squeezes and expands in the direction of wave propagation, like sound in air. *Secondary waves* (S-waves) are shear waves: the rock moves perpendicular to the propagation direction, like a rope being shaken. Both types travel at speeds determined by the elastic constants of the rock and its density. P-waves travel faster:

$$v_P = \sqrt{\frac{K + \frac{4}{3}G}{\rho}} \quad (8.1)$$

$$v_S = \sqrt{\frac{G}{\rho}} \quad (8.2)$$

In these equations,  $K$  is the bulk modulus (resistance to volumetric compression),  $G$  is the shear modulus, and  $\rho$  is the density. Both  $K$  and  $G$  are derived from Young's Modulus and Poisson's ratio — they are different combinations of the same two material constants that Cauchy identified as sufficient for an isotropic elastic solid.

Seismologists read the time delay between the arrival of P-waves and S-waves at a station to estimate the distance to the earthquake's epicenter. The ratio of wave speeds depends on Poisson's ratio alone:  $v_P/v_S = \sqrt{2(1-\nu)/(1-2\nu)}$ . By monitoring how this ratio changes in regions of active tectonics, researchers can detect changes in the state of stress in the crust before a rupture occurs. The entire field of seismological monitoring — including the efforts to predict earthquakes — rests on Hooke's Law applied to rock.

### 8.3 Piezoelectricity: Stress Becomes Voltage

In 1880, Pierre and Jacques Curie discovered that compressing a quartz crystal along certain directions produced an electric charge on its surfaces. The effect — named piezoelectricity, from the Greek *piezein*, to press — was understood, within a few years, to be a direct consequence of the crystal structure: the mechanical deformation shifted positive and negative ions relative to each other, generating a net electric dipole moment. Stress, via strain, produced polarization.

The constitutive equations for a piezoelectric material are an extension of Hooke's Law. For a simple linear piezoelectric:

$$D_i = d_{ijk}\sigma_{jk} + \varepsilon_{ij}^\sigma E_j \quad (8.3)$$

where  $D_i$  is the electric displacement (related to charge),  $\sigma_{jk}$  is the stress tensor,  $E_j$  is the electric field, and  $d_{ijk}$  is the piezoelectric coefficient tensor. The first term is the mechanical-to-electrical coupling: stress produces charge. The second is the dielectric contribution. The mechanical part,  $d_{ijk}\sigma_{jk}$ , is linear in stress — it is Hooke's Law with an electrical output.

The converse effect also exists: apply a voltage, and the crystal deforms. This converse piezoelectric effect is used in ultrasound transducers, inkjet printer heads, active vibration

control systems, and the piezoelectric actuators that position the read heads in hard disk drives with nanometer precision.

The quartz oscillator in your watch — and in virtually every digital clock made in the last fifty years — uses the resonant frequency of a quartz crystal. The crystal is cut so that it vibrates at a precise frequency, governed by its elastic constants and geometry, when electrically excited. Those elastic constants are Young’s Modulus and the shear modulus of quartz, measured at the crystal lattice level. Every tick of a quartz clock is Hooke’s Law operating at the microscale.

## 8.4 Finite Element Analysis: The World Assembled from Springs

Modern structural engineering, automotive design, aerospace analysis, and biomedical device development all rely on a computational technique called *finite element analysis*, or FEA. The technique has been refined enormously since its mathematical foundations were laid in the 1950s and 1960s, but its conceptual core is simple: a complex structure is divided into a large number of small, simple pieces called *elements*, each of which is assumed to behave as a linear elastic solid obeying Hooke’s Law, and the behavior of the whole is assembled from the behaviors of the parts.

The assembly process — converting element-level stress-strain relationships into a system-level stiffness matrix — is essentially the same operation that a structural engineer performs by hand for simple frameworks: the spring constants of individual elements are combined into a global stiffness matrix  $\mathbf{K}$ , and the displacements  $\mathbf{u}$  of all nodes are found by solving:

$$\mathbf{K}\mathbf{u} = \mathbf{F} \tag{8.4}$$

This is Hooke’s Law,  $F = kx$ , written in matrix form for a system with thousands or millions of degrees of freedom simultaneously. The global stiffness matrix  $\mathbf{K}$  encodes all the elastic relationships of all the elements;  $\mathbf{F}$  is the vector of applied forces;  $\mathbf{u}$  is the vector of nodal displacements that Hooke’s Law requires.

A commercial aircraft wing, analyzed before first flight, is represented in FEA by a mesh of perhaps ten million elements. Each element is described by its material’s Young’s Modulus and Poisson’s ratio. The stiffness matrix for ten million elements contains trillions of potential entries, though most are zero because each element interacts only with its immediate neighbors. The solution — the displacement of every node in the mesh under every critical loading condition — is found by matrix factorization algorithms running on computing clusters. The result tells engineers where the wing deflects, where the stresses are highest, and where the design margins are smallest.

The wing has no spring constant in the sense Hooke meant. But the mathematics underlying its analysis is  $F = kx$ , extended by three and a half centuries of mathematical development into a form Hooke would recognize in principle and be astonished by in scale.

## 8.5 MEMS: Hooke’s Law at the Micron Scale

Inside your smartphone is an accelerometer. It is a silicon device etched to micron tolerances using the same photolithographic processes that make computer chips. At its heart is a tiny proof mass — a small silicon slab, microns in dimension — suspended by silicon springs. When

the phone accelerates, the proof mass lags behind due to its inertia, deflecting the springs that suspend it. The deflection is measured electrically via capacitance changes, and the measured deflection, multiplied by the known spring constant of the suspension, gives the force, which divided by the known mass gives the acceleration. The device is, at the most basic level, Hooke's experiment in miniature.

These *microelectromechanical systems* (MEMS) accelerometers depend on Hooke's Law being accurate at the micron scale. It is. The silicon springs are beams, and their spring constants are calculated using the Euler-Bernoulli beam equations that we saw in the previous chapter. Young's Modulus for single-crystal silicon is approximately 130 gigapascals — well characterized, highly consistent, and applicable all the way down to the scale of individual crystal unit cells. Hooke's Law does not weaken at small scales. It was always a law about atomic bond stiffness, and atomic bonds do not change their character as the structure around them shrinks.

The MEMS accelerometer in your phone also governs the airbag system in your car, the motion sensing in game controllers, and the attitude control systems in satellites. In each case, the principle is the same: a spring deflects under a force, and the deflection is proportional to the force. Hooke, hanging weights from a spring in a London laboratory, would have recognized the experiment immediately, had he survived to see the scale at which it is now routinely performed.

## 8.6 Why Everything Is a Spring

A natural question at the end of this survey: why does Hooke's Law work so broadly? Why does it describe not just metal springs but rock, quartz, silicon, bone, steel, glass, and wood?

The answer is one of the most useful ideas in all of physics, and it goes by the name of *perturbation theory* or, in this context, *small oscillations theory*.

Almost any physical system in equilibrium — atoms in a crystal, a pendulum at rest, a ship at anchor — can be described by some potential energy function  $U(x)$ , where  $x$  is the relevant displacement from equilibrium. At equilibrium, the potential energy is at a minimum:  $dU/dx = 0$  at  $x = 0$ . Now expand the potential energy in a Taylor series around this minimum:

$$U(x) = U(0) + \left. \frac{dU}{dx} \right|_0 x + \frac{1}{2} \left. \frac{d^2U}{dx^2} \right|_0 x^2 + \dots \quad (8.5)$$

The first term is a constant, irrelevant to forces. The second term is zero, because we are at a minimum. The third term — proportional to  $x^2$  — is the first nontrivial term for small displacements  $x$ . The force derived from this potential is:

$$F = -\frac{dU}{dx} \approx -\left. \frac{d^2U}{dx^2} \right|_0 x = -kx \quad (8.6)$$

This is Hooke's Law, with spring constant  $k = d^2U/dx^2$  evaluated at equilibrium. It is not a special property of springs. It is a mathematical inevitability: any smooth potential energy function looks linear in force near its minimum, for small enough displacements. Every material, every system in equilibrium, is approximately Hookean at small perturbations. The law is universal not because springs are special, but because equilibrium is common and small displacements are everywhere.

The question of whether Hooke's Law applies to a given system is therefore not a question about the material. It is a question about the scale of the deformation. For deformations small relative to the atomic spacing — as is the case for virtually all structural applications, all seismic waves, all piezoelectric effects, and all MEMS devices — the Taylor series truncates at the linear term and Hooke's Law holds exactly. For deformations that are not small — rubber under large extension, biological tissue through its full range of motion, a metal past its yield point — higher-order terms become significant and Hooke's Law fails.

The law's extraordinary breadth of application is, in the end, a consequence of a beautiful and simple mathematical fact: small displacements from equilibrium are always linear, for any smooth potential, no matter what the underlying physics. Hooke did not know this. He measured springs. But what he found was something far more general: the universal behavior of matter at small deformations, encoded in the simplest possible equation a proportionality between cause and effect.

*Ut tensio, sic vis.* As the extension, so the force.

Three hundred and fifty years on, it still is.



## Chapter 9

# References



# Appendix A

## Python Code Reference

This appendix collects the Python functions used throughout the book to generate figures and numerical examples. All code runs inside the project's `pixi` environment. To reproduce any figure, run `pixi run render` from the project root.

### A.1 Spring Force Functions

### A.2 Beam Deflection Functions

### A.3 Stress-Strain Functions

### A.4 Griffith Fracture

### A.5 Seismic Wave Speeds

